



NUMERICAL ANALYSIS OF STRESS AND TEMPERATURE IN THE FRICTION STIR WELDING (FSW) PROCESS OF STEEL

NUMERYCZNA ANALIZA ROZKŁADU NAPRĘŻEŃ I TEMPERATURY W PROCESIE ZGRZEWANIA TARCIOWEGO Z PRZEMIESZANIEM DLA STALI

MICHAŁ SZCZECINA*
Kielce University of Technology, Poland

Abstract

Friction stir welding (FSW) is a modern technology for joining various metals, which has already undergone many laboratory tests, but still requires the development of numerical models. Author of the paper decided to summarize the current state of scientific knowledge regarding the modelling of the FSW process using the finite element method (FEM) and showed the main directions of development of numerical research on this process. Very advanced models are a combination of solid mechanics and fluid dynamics, but they often require expanding the computing environment with its own subroutines, as well as calibration and validation of some material parameter and constants occurring e.g. in the heat generation and heat flow laws. The Author of the paper proposed his own, simplified model, based on the computational solid mechanics and Lagrangian formulation. The model turned out to be an effective tool to reproduce stress and temperature fields during the FSW process.

Keywords: friction stir welding, numerical modelling, Abaqus, FEMs

Streszczenie

Zgrzewanie tarciove z przemieszaniem (FSW) jest nowoczesną technologią łączenia różnych metali, posiadającą wiele zalet w porównaniu z tradycyjnym spawaniem. Zgrzewanie tarciove zostało do tej pory poddane licznym badaniom laboratoryjnym, natomiast wymaga ciągłego rozwoju modeli numerycznych do symulacji tego procesu metodą elementów skończonych (MES). Autor artykułu postanowił dokonać podsumowania aktualnego stanu wiedzy dotyczącej modelowania zgrzewania tarciowego przy użyciu MES oraz wskazać główne kierunki rozwoju symulacji numerycznych tego procesu. Zaawansowane modele numeryczne zgrzewania tarciowego są kombinacją mechaniki ciała stałego z dynamiką płynów, a więc często wymagają rozbudowania środowiska obliczeniowego za pomocą własnych podprogramów, jak również kalibracji i walidacji wielu parametrów i stałych wymaganych do zdefiniowania np. prawa wytwarzania ciepła i prawa przepływu strumienia ciepła. Autor zaproponował swój własny uproszczony model bazujący na mechanice ciała stałego i opisie Lagrange'a. Model okazał się efektywnym narzędziem do odtworzenia naprężeń i pola temperatury w procesie zgrzewania tarciowego z przemieszaniem.

Słowa kluczowe: zgrzewanie tarciove z przemieszaniem, modelowanie numeryczne, Abaqus, MES

1. INTRODUCTION

Friction stir welding (FSW) was invented by Thomas et al. [1] in 1991, so one can say that it is a relatively young and innovative technology. For over 30 years FWS has been used in many high technology applications, e.g. aerospace [2], and has also been a subject of numerous scientific research. There are a few advantages of the FSW process, and namely: lower temperatures in comparison with traditional welding, no melting, a FSW tool is unconsumable [3]. A development of computational methods, especially finite element method (FEM), allowed for advanced numerical modelling of the stir welding process. The modelling covers not only mechanical, but also thermal and coupled thermal-mechanical behaviour of welded parts. Some important phenomena can also be taken into account, i.e. heat generation and dissipation, metal flow, sticking and sliding. Author of the paper summarized the current state of the research field, indicate some issues of the numerical modelling of the SFW and propose their

own, simplified numerical approach of the process. Additionally, author demonstrate how the FSW can be modelled using Abaqus [4] software to reproduce stress and temperature fields during the process.

1.1. General description of the process

The FSW process can be divided into three phases: plunging, dwelling and welding. An operating tool consists of a conical shoulder and a pin, which is shaped in a few various ways, e.g. cylindrical, threaded, square or tapered (Fig. 1). In the first stage the rotating pin is lowered down into a joint line, which is determined by two surfaces (workpieces) prepared to be welded. Then the rotating tool is held steady to produce heat, which dissipates into the neighbouring material. In this stage temperature of the welded parts increases which causes material softening. Finally, the welding operation is performed by the relative displacement of the tool along the joint line (Fig. 2). The material transported from the front to the back of the welding tool forms a joint between two workpieces.

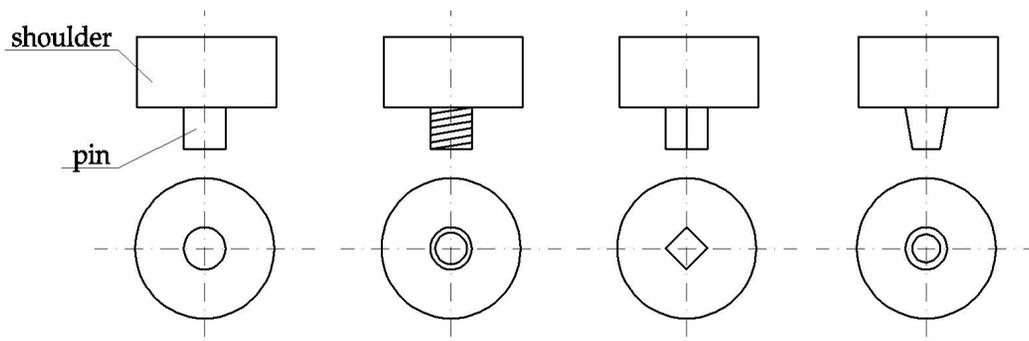


Fig. 1. Sample shapes of the operating tool used in the FSW process

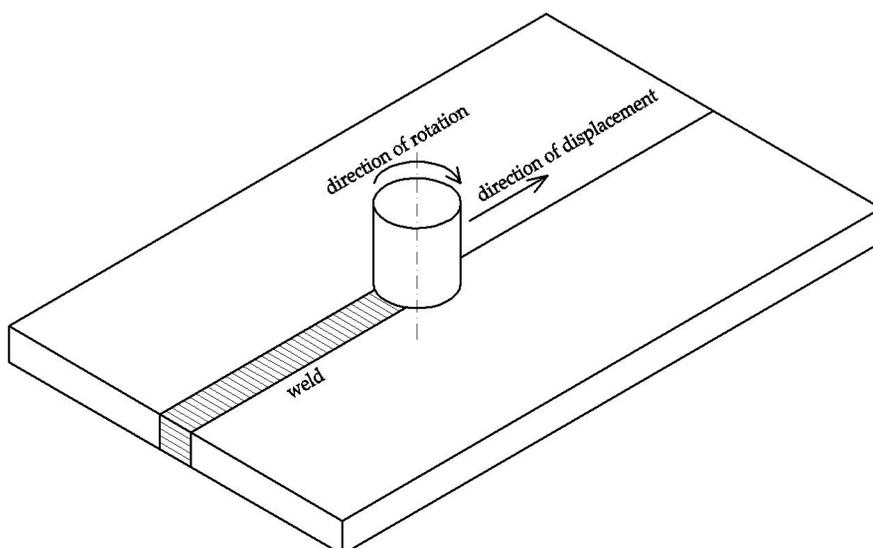


Fig. 2. General view of the FSW process

There are many important parameters that affect the whole welding process, first of all:

- geometry of the whole operating tool, discussed inter alia by Mishra&Ma [5], Sun et al. [6] and Nandan et al. [7];
- geometry of the operating tool shoulder [8];
- tilt of the operating tool and target depth [5];
- speed (transverse and rotational) of the operating tool [5, 6, 9];
- downward force;
- metallurgical and microstructural aspects, grain size and microhardness [6, 10];
- welding configuration of workpieces made of dissimilar metals [6];
- preheating or cooling of workpieces [5];
- joint design, especially when the joint is more complicated than a simple butt or lap joint [5].

A few important phenomena which occur in the whole FSW process and a few properties of welded materials have also been described in the subject literature [5]:

- metal flow, visualized in laboratory tests with the use of tracer technique by marker, welding of dissimilar materials, microstructural observations or particle tracing [11];
- heat generation, heat transfer [12];
- microstructural evolution in a nugget zone [13], a thermo-mechanically affected zone and a heat-affected zone [14];
- change in hardness of alloys [15];
- residual stress [16];
- change in postweld mechanical properties, especially strength, ductility, fracture toughness and fatigue [5];
- corrosion behaviour [17];
- interfacial sticking and slipping [18].

Kossakowski et al. [19] performed a macrostructural analysis of FSW joints, taking into consideration weld structure, rotation speed, tool travel and showing joint defects as a result of process parameters. The same authors [20] analysed effect of tool rotation and travel speed on joint parameters. Static and dynamic performance of FSW butt joint, including fatigue crack growth, was presented by Richards [21].

According to Mishra [5], the FSW process has many benefits, some of them in comparison to a traditional welding. The benefits can be divided into metallurgical (no cracking of a weld, no loss of material, low distortion of workpieces etc.), environmental (no grinding waste, no use of shielding gas, surface cleaning or solvents for degreasing) and energy benefits.

1.2. Numerical modeling of the FSW process

A finite element method approach to the FSW process has already been applied in many scientific research. A literature review on the topic was presented by Neto&Neto [3] and a comparative study of different FEM approaches can be found in a work of Meyghani et al. [22]. A critical review of the FSW numerical modelling was given by Lorrain et al. [23]. Some important introduction to FEM simulation of the FSW was presented in a few PhD and master theses [24-26]. There are few current studies and literature reviews concerning current progress in numerical modelling; one can refer to works of Bhattacharjee&Biswas [27] and Sen&Murugesan [28]. Author of this paper decided to complete and summarize the current state of the research field.

The first issue of a proper numerical simulation of the FSW process is a choice of theory that describes the material flow and thermo-mechanical behaviour. There are two main theories chosen for this purpose: computational fluid dynamics (CFD) and computational solid mechanics (CSM). A detailed study of both approaches was given by Bhattacharjee&Biswas [27]. Pros and cons of the above mentioned theories were described by Meyghani et al. [29].

The CFD is based on the Eulerian description with a fixed mesh and material is assumed as a non-Newtonian fluid. This approach allows to avoid numerical problems in case of large strains and element distortions, but it does not allow the separation of the element. Thanks to the flow boundary conditions only a small region around the welding tool can be modelled, which leads to a significant reduction of finite elements number and calculation time. Colegrove&Shercliff [30, 31] demonstrated the use of the CFD package FLUENT and modelled the 3D metal flow. The modelled recreated the FSW process quite well, but over-predicted the weld temperature and poorly predicted the welding forces. A viscosity relationship, including material softening, was proposed to overcome those problems. Jacquin et al. [32] presented a simple 3D thermo-mechanical model with velocity fields in a steady-state, solved using Abaqus/CAE software with extra procedures written in Fortran.

The CSM approach is based on the Lagrangian formulation, where the finite element mesh is attached to the material and follows its deformation [29]. Using an explicit time integration one can avoid numerical problems with convergence, but still large

distortions can lead to a significant reduction of a time step size. A detailed description and discussion on a global and local level modelling is described in a monograph written by Dialami et al. [33]. The same authors [34] demonstrated the finite volume method to model the FSW process. The technique of tracer particles in the FEM model based on CSM was presented by Gao and co-workers [35] in Abaqus environment.

The geometrical interpretation of the main difference between the Lagrangian and Eulerian formulation is presented in Figure 3. An excessive distortion of the FE mesh can appear in the Lagrangian description while in the Eulerian element boundaries do not have to coincide with the mesh.

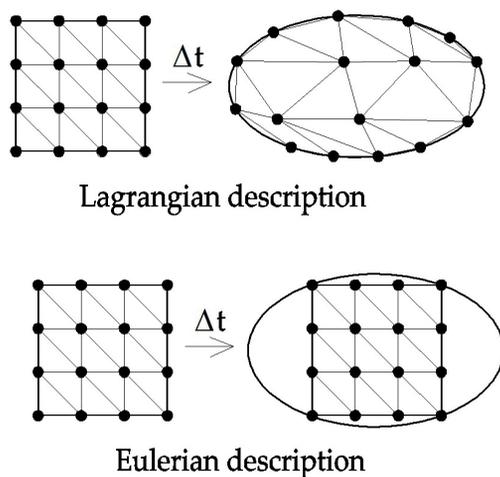


Fig. 3. Different mesh behavior in the Lagrangian and Eulerian formulation

In the research field there are also known and developed mixed methods, which are combinations of both Lagrangian and Eulerian formulations, and namely the so-called Arbitrary Lagrangian-Eulerian (ALE) and Coupled Eulerian-Lagrangian (CEL). The first approach assumes splitting into subsequent Eulerian and Lagrangian steps, computing the mesh velocity and remapping [25, 36]. The CEL method allows to define the workpiece as an Eulerian body while the welding tool is modelled as a rigid Lagrangian body [37]. This approach can bring promising results, as shown in the works [38, 39].

Another problem related to the FEM simulation of the FSW is a proper definition of the thermal and thermo-mechanical model. An early work of Chen&Kovacevic [40] presents the use of the ANSYS software to reproduce the influence of thermal fluctuations on the stress tensor components. One year later, Schmidt&Hattel

[41] proposed a fully coupled thermomechanical 3D model using Abaqus software and the ALE formulation. The same authors [42] described basics of the thermal modelling in the FSW process, including the heat conduction equation and the total heat generation equation and presented a new thermal-pseudomechanical model. Hamilton and co-workers [43] presented a thermal model introducing a slip factor, determined according to energy per unit length of weld. Mehta et al. [44] defined subroutines in Abaqus environment called DFLUX, FILM, USDFLD and UMASFL to capture the non-uniform heat flux, convective heat transfer, tool properties and mass velocity. Their work concerned the welding tools with polygonal pins.

Coupled thermo-mechanical models are nowadays widely used in the FSW simulations. Chiumenti et al. [45] presented a fully coupled model, where the local form of the FSW problem is stated in the form of coupled mechanical and transient heat transfer equations. It is also possible to apply a thermo-mechanically coupled viscoplastic flow model. A sample formulation is shown in a work of Santiago et al. [46], inspired with an early work of Ulysse [47].

Another important issue of the FSW simulation is a formulation of contact conditions, including sliding and sticking. Usually researchers define a classical Coulomb friction law, which can be expressed in the form:

$$\tau = \mu P \quad (1)$$

where: τ – indicates the friction stress (in [Pa]), μ – friction coefficient (unitless), P – contact force (in [N]).

Zhang [48] compared the classical and modified Coulomb laws and stated, that the classical law is limited to lower angular velocity of the welding tool. In case of higher velocity, the modified Coulomb law should be applied. More complex contact states were categorized and presented by Schmidt et al. [49] and contact conditions can be defined separately for sticking and sliding [3].

Finally, a constitutive material model for plastic or viscoplastic behaviour and behaviour for flow modelling of metal is also required as an input. Ulysse [47] and Santiago et al. [46] modelled the FSW process using 3D viscoplastic modelling. The other approach was presented by Zhu&Chao [50], which applied the von Mises yield criterion and the associated flow rule. As regards materials behaviour for flow modelling, the Sellars-Tegart and John-Cook models are the most common approach [3].

The finite element method is the most common technique used in numerical modelling of the FSW process. Sen&Murugesan [28] presented a comparative study of three FEM software packages, and namely ANSYS, ABAQUS and FLUENT. A very interesting approach to the FSW modelling is a use of artificial neural networks (ANN). Abdullah et al. [51] showed how to use ANN to model surface roughness of aluminium alloy. Okuyucu et al. [52] analysed a correlation between the FSW parameters and mechanical properties of aluminium using ANN.

One of the most important issues in welding is a phase transition in material. The phenomenon was not yet taken into consideration by the author of the paper because of its complexity. Nevertheless, a few publications [53-55] on the issue show that FEM simulation of the effect is possible, among other using SYSWELD software.

To sum up, all the above presented approaches are based on very robust, advanced numerical models. The models demand on a precise calibration and validation of input data and can take into account various physical phenomena, accompanying the FSW process. Numerical modelling of the FSW is developing still and reported results of FEM calculations are still closer to laboratory tests results. On the other hand, structural engineers need also simple numerical models which can give quick information about stress and temperature fields during the FSW process. Author of the paper made an attempt to create such a model, presented in the following sections.

2. INPUT DATA

The author of the paper would like to demonstrate a use of a simple numerical model of the FSW process, defined in the Abaqus environment [4]. The main goal of the research is a numerical reproduction of residual stress after the FSW process. Geometry and material properties of specimens were taken from the laboratory experiment of Hashemzadeh et al. [56]. All materials were defined as isotropic. Two steel plates, each 2000x150x12 mm of dimensions, were modelled. Mechanical and thermal properties of ASTM A36 carbon steel applied in the model are listed in the Table 1. Temperature dependency of the properties were taken from the paper of Chang&Teng [57]. Ultimate strength of steel was assumed as 450 MPa [58] and Poisson’s ratio was temperature-independent and equal to 0.3. The author’s choice fell on steel, because the author of the paper recognized that laboratory tests on steel [56] were documented in the most detailed way among all the results of laboratory tests that were collected for the purpose of performing numerical analyses in this paper.

The FSW tool was assumed as hybrid W-Re/pcBN [51]; geometry of the tool was defined according to [59] and shown in Figure 4. Main properties of the FSW process are as listed: rotation speed of the tool: 150 rpm, transverse speed of the tool: 0.1 ms⁻¹, vertical force acting on the shoulder of the tool: 90 kN [56]. Friction coefficient was assumed as equal to 0.3.

Table 1. Mechanical and thermal properties of ASTM A36 carbon steel

| Temperature [°C] | Young’s modulus [GPa] | Yield stress [MPa] | Specific heat [Jkg ⁻¹ K ⁻¹] | Conductivity [Wm ⁻¹ K ⁻¹] | Expansion [10 ⁻⁶ K ⁻¹] |
|------------------|-----------------------|--------------------|--|--|---|
| 20 | 210 | 380 | 450 | 51 | 11.2 |
| 100 | 195 | 340 | 475 | 50 | 11.8 |
| 210 | 195 | 320 | 530 | 49 | 12.4 |
| 330 | 185 | 262 | 560 | 46 | 13.1 |
| 420 | 168 | 190 | 630 | 41 | 13.6 |
| 540 | 118 | 145 | 720 | 38 | 14.1 |
| 660 | 52 | 75 | 830 | 34 | 14.6 |
| 780 | 12 | 40 | 910 | 28 | 14.6 |
| 985 | 11.8 | 38 | 1055 | 25 | 14.6 |
| 1320 | 10.4 | 28 | 2000 | 32 | 14.6 |
| 1420 | 10.2 | 25 | 2100 | 42 | 14.6 |
| 1500 | 10 | 20 | 2150 | 42 | 14.6 |

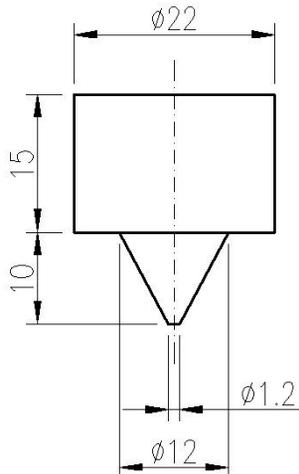


Fig. 4. Geometry of the welding tool (dimensions in [mm])

3. FEM MODEL

Numerical analysis was performed using the finite element method in Abaqus environment [4]. Author decided to apply the Lagrangian approach and the explicit dynamic step with the fully coupled thermal-stress analysis (the so-called temperature-displacement

in the Abaqus code). The plates were meshed with C3D8T brick finite elements, 12x12x3 mm each and the welding tool was discretized with C3D4T tetrahedrons and the average mesh size was assumed as 3 mm. The meshed assembly is presented in Figure 5 while, for the sake of visibility, the magnified welding tool with the plates were shown in Figure 6.

All mechanical and thermal properties were defined as shown in the section 2. Moreover, the classical von Mises yield criterion (input variable PLASTICITY in Abaqus code) was taken into account. The stress-strain relationship was defined as bi-linear with the following values of the yield strain (Eq. 2) and ultimate strain (Eq. 3) in the initial temperature, equal to 20°C:

$$\epsilon_y = \frac{\sigma_y}{E} = \frac{380 \text{ MPa}}{210 \text{ GPa}} = 0.00181 \quad (2)$$

$$\epsilon_u = \frac{\sigma_u}{E} = \frac{450 \text{ MPa}}{210 \text{ GPa}} = 0.00214 \quad (3)$$

and the relationship of a uniaxial behaviour is presented in Figure 7.

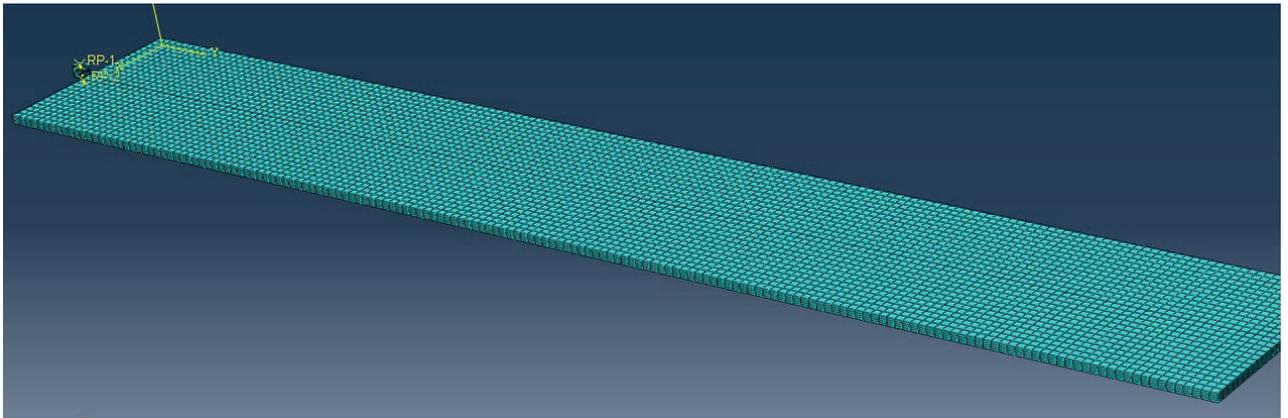


Fig. 5. Meshing of the plates and the welding tool

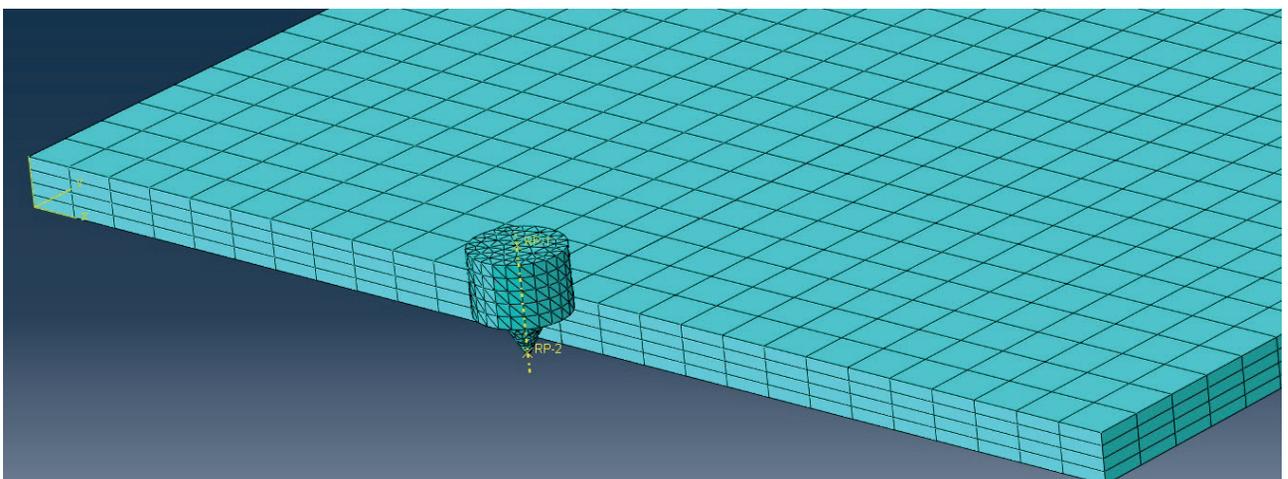


Fig. 6. The magnified welding tool with the edges of the plates

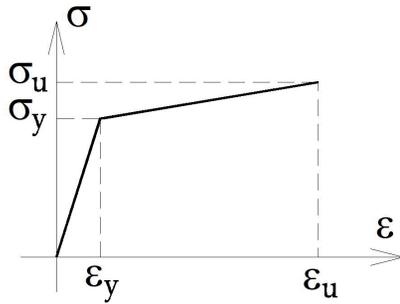


Fig. 7. Stress-strain curve defined in the numerical model

The welding tool was defined as a rigid body rotating with a constant angular velocity and the plates – as deformable elements moving with a constant transverse velocity. Boundary conditions were defined as follows:

- transverse velocity of the plates, shown in Figure 8;
- angular velocity of the welding tool and force acting on the shoulder, shown in Figure 9.

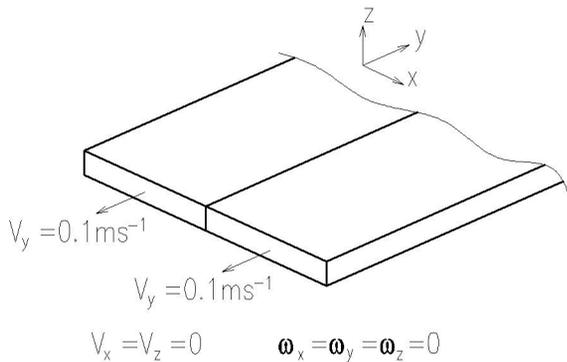


Fig. 8. Boundary conditions – transverse velocity of the welded plate

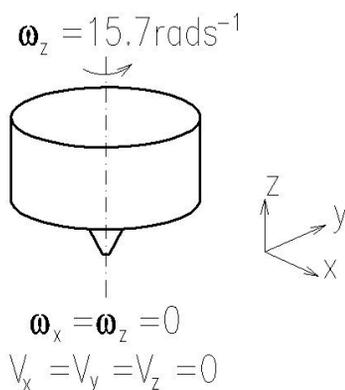


Fig. 9. Boundary conditions – angular velocity of the welding tool

Ambient temperature was set as 20°C, using the “predefined fields” option in the initial step in all nodes of the meshed model. A surface-to-surface contact between the welding tool and the plates was defined as follows:

- tangential behaviour – friction coefficient 0.3 (called “penalty” in Abaqus code);
- normal behaviour – “hard contact”;
- heat generation with default 0.5 fraction of converted heat distributed to slave surface.

A time period was set as 22 s, which is enough for the welding tool to make a full weld joining the plates and to come out of them. A type of incrementation was assumed as fixed with the user-defined time increment equal to 0.001.

4. ANALYSIS OF TEMPERATURE AND STRESS IN PLATES

In order to visualize the whole welding process and stress during the welding process, the author decided to present maximal principal stress and nodal temperatures. Maximal principal stress on the upper surface of the welded plates after time $t = 22 \text{ s}$ is presented in Figure 10 (please note, that the results are presented in [Pa] using a native notation in Abaqus code, e.g. e+08 means 10 to the power of 8). We can see how the stress field is propagating on the surface of the plates while the welding tool is moving along the Y axis. Moreover, the stress field expands also in the X axis, transverse to the welding direction. Figures 11 and 12 present the maximal principal stress in a transverse cross section and nodal temperature in a longitudinal cross section of the weld. Temperatures obtained under the welding tool are above the steel melting point and stress exceeds the yield stress in three columns (in each plate) of finite elements, which proves a limited range of material deformation caused by the process. Moreover, one can see that the stress and temperature distribution in the left-hand side and right-hand side plates (looking from viewport adopted in Fig. 11) are different. This fact can be explained by rotational motion of the welding tool. Because of the motion, in one of the plates metal particles are displaced in front of the leading edge of the tool shoulder while in the other plate they are moved to the trailing edge, so the whole model is not symmetrical.

Variation of temperature and principal stress depending on the distance z from the top of the right-hand side (see Fig. 11) plate in the section under the welding tool are presented in Figures 13 and 14. High values of temperature (in some nodes above the melting point) and stress indicate large yielding of steel in the weld zone during the process, which is necessary to stir metal particles of both welded plates.

Total strains and plastic strains (maximal principal) are shown in Figures 15 and 16. The values of the strains are lower than ultimate strains presented in the previous chapter.

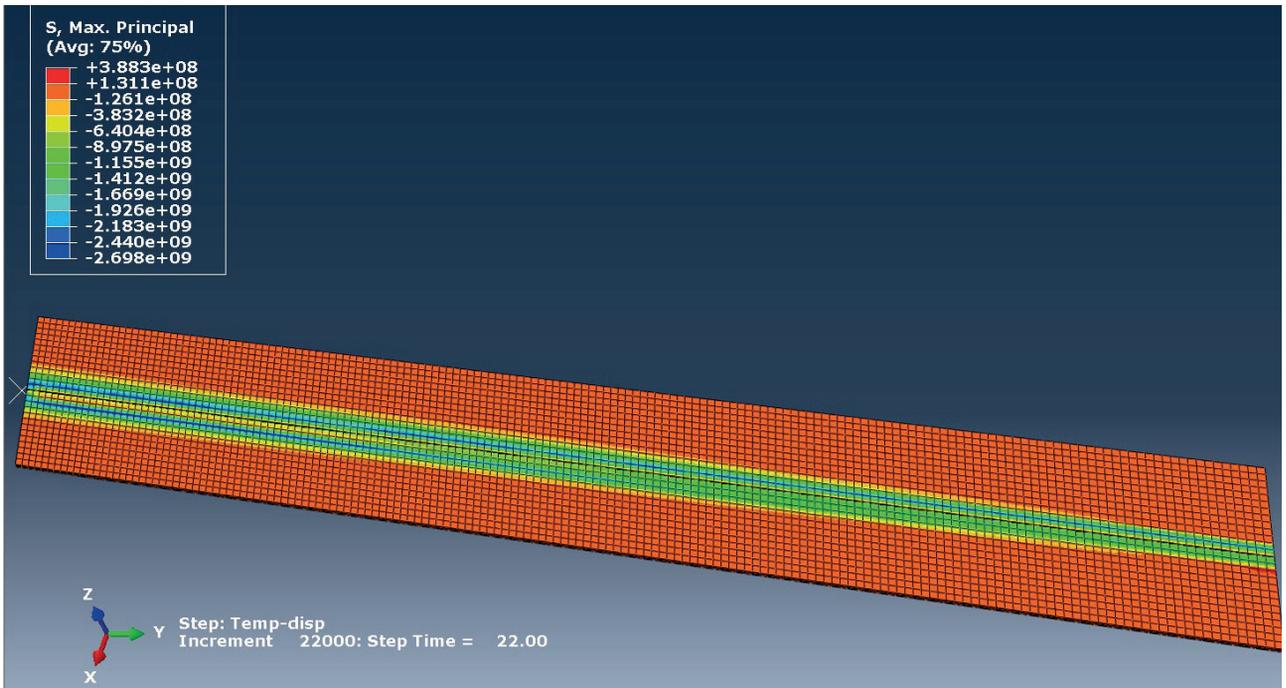


Fig. 10. Maximal principal stress after time $t = 22.0$ s; all results expressed in Pa

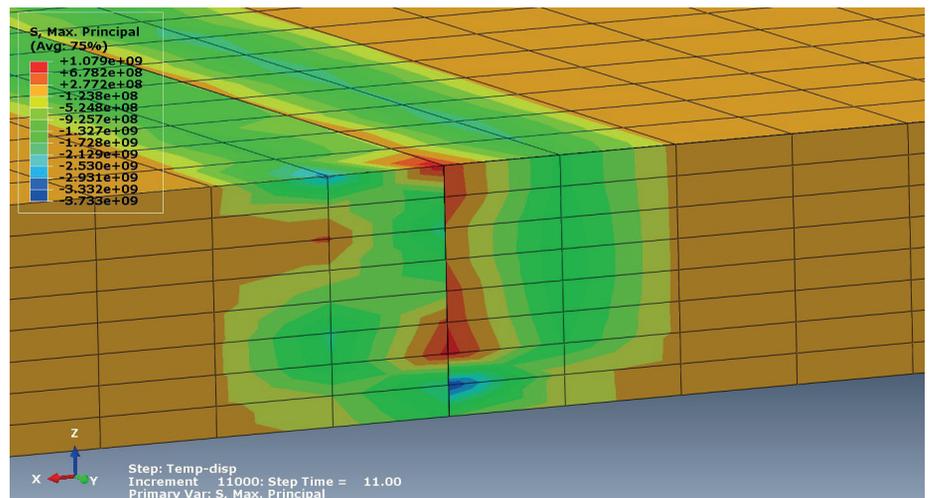


Fig. 11. Maximal principal stress in a transverse cross section of the plates, $t = 11.0$ s

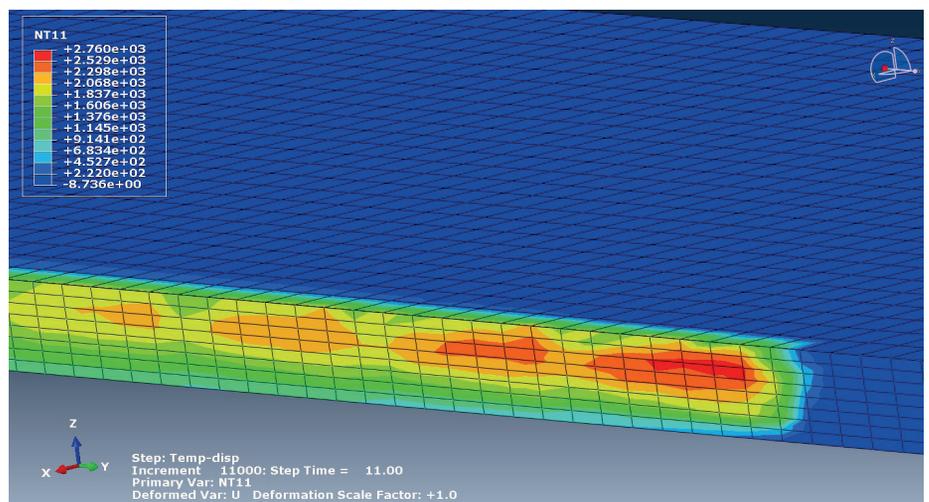


Fig. 12. Nodal temperatures (in $^{\circ}\text{C}$) in a longitudinal cross section of the weld, $t = 11.0$ s

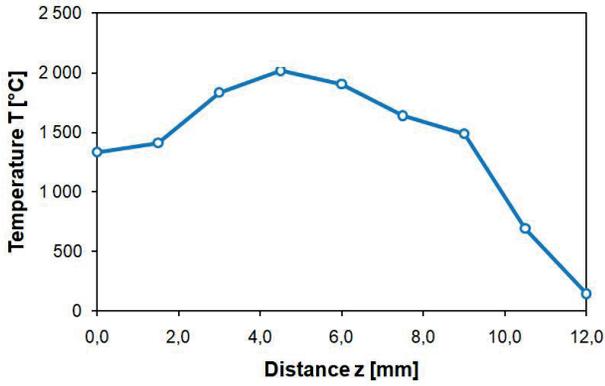


Fig. 13. Nodal temperature in a transverse section of the right-hand side plate vs distance z from the top of the plate, $t = 11.0$ s

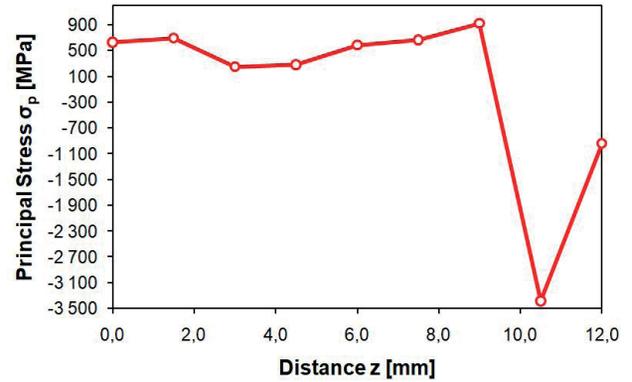


Fig. 14. Principal stress in a transverse section of the right-hand side plate vs distance z from the top of the plate, $t = 11.0$ s

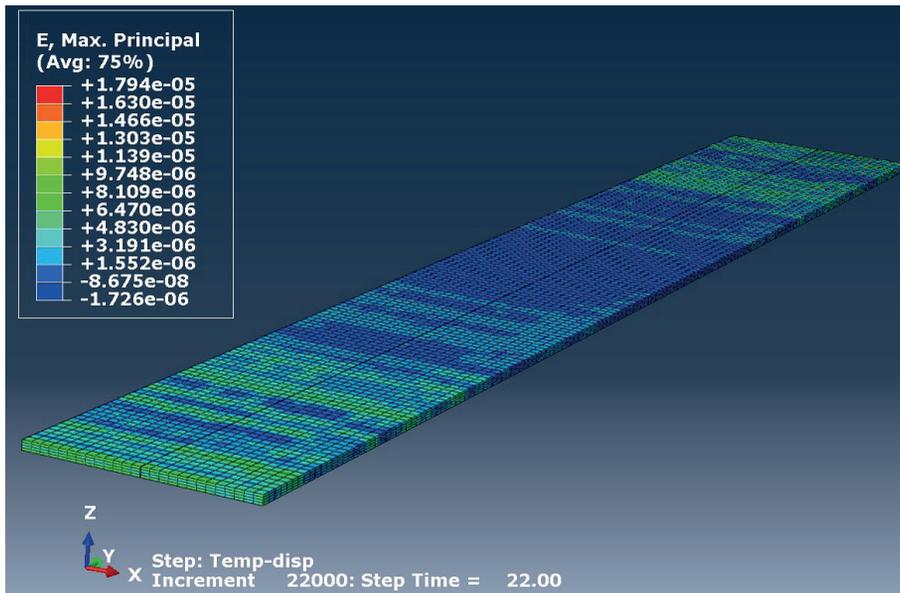


Fig. 15. Maximal principal total strains after time $t = 22.0$ s

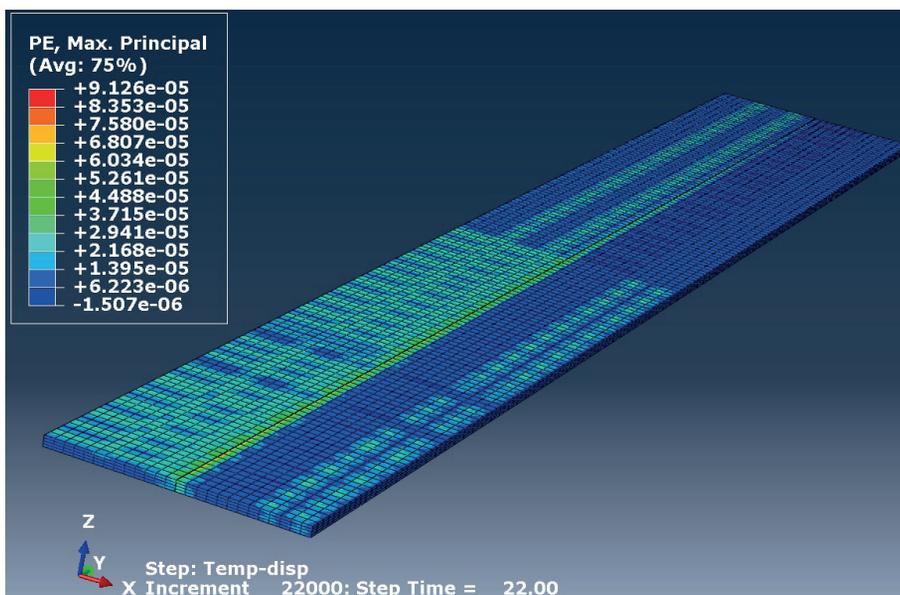


Fig. 16. Maximal principal plastic strains after time $t = 22.0$ s

5. DISCUSSION AND CONCLUSIONS

According to Neto&Neto [3], experimental investigations of the FSW process focused mainly on adjusting parameters such as angular and transverse speed of the welding tool, geometry of the tool, depth of penetration in welded parts. On the other hand, numerical simulation can be used to visualize some phenomena and results easier than in experimental investigations, e.g. material flow, heat generation and transfer, residual stress field and so on. Ten years after Neto&Neto [3] publication, the need for a new review and summary of progress in the numerical simulation of the FSW arose. This paper is an attempt to fill the gap. Author of this paper presented also a simple FEM simulation based on the Lagrangian approach and showed how to estimate stress in welded parts. The simple model can be useful in a practical use, e.g. for a structural engineer when designing a welded joint to make sure how large temperature and stress can be during welding.

The numerical model can be an effective tool in optimization and controlling of the FSW process. For a thorough scientific analysis, especially when

laboratory tests are compared with FEM simulations, more robust and complex numerical models are necessary. The most promising approaches are the Coupled Eulerian-Lagrangian (CEL) and Arbitrary Lagrangian-Eulerian (ALE) approach. Both methods are still being developed and while using them it is also possible to take into account the thermo-mechanical model of the FSW process. Researchers have already recognized the capabilities of advanced FEM codes and also try to incorporate their own subroutines. Some new opportunities are created by the use of artificial neural networks (ANN). At the moment ANN are rather applied to model some particular parameters of the FSW process or to find correlation between the parameters. The next step can be the so-called hybrid FEM and ANN modelling.

In this paper only one particular type of carbon steel was examined. As it was shown in works of Kossakowski et al. [60, 61], aluminium alloys and stainless steel are also in the scope of interest of contemporary construction industry. The author of this paper plans also to investigate the FSW process of these materials.

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